#### 4.3 Yield Criteria

- Cannot perform tests for all combinations of 3D loading so we need yield criterion to generalize from small number of tests.
- What are the ideal properties of a 3D yield criterion?

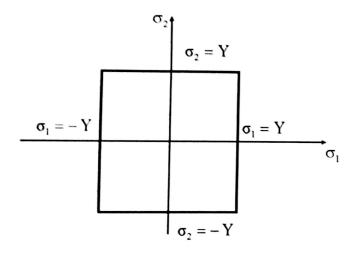
 $f(\sigma_{ij}, Y) < 0$  Elastic Behavior  $f(\sigma_{ij}, Y) = 0$  Onset of Inelastic Behavior

- We usually visualize yield criterion by a surface in principal stress space. Why?
- We also calculate "effective" stress to compare with yield stress.

### Maximum principal stress criterion William Rankine (1820-1872)

Applicable to brittle materials (mostly in tension)

$$f = \max(|\sigma_1|, |\sigma_2|, |\sigma_3|) = Y$$



Maximum Principal Stress Yield Surface

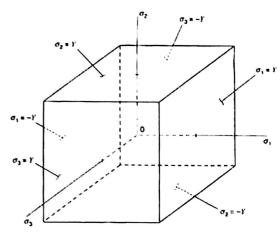


Figure 4.7 Maximum principal stress yield surface.

#### Maximum principal strain criterion Adhémar Jean Claude Barré de Saint-Venant 1797 - 1886

- Has the advantage that strains are often easier to measure than stresses
- Assume that epsilon1 is the largest principal strain

$$\varepsilon_{1} = \frac{1}{E} (\sigma_{1} - v\sigma_{2} - v\sigma_{3})$$

$$f_{1} = [\sigma_{1} - v\sigma_{2} - v\sigma_{3}] - Y$$

$$\sigma_{1} - v\sigma_{2} - v\sigma_{3} = \pm Y$$

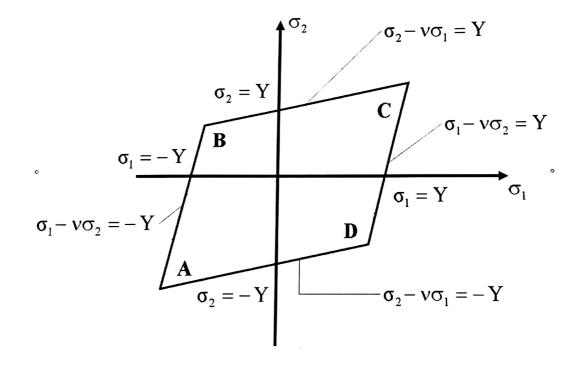
$$\sigma_{e} = \max_{i \neq i \neq k} |\sigma_{i} - v\sigma_{j} - v\sigma_{k}| \rightarrow f = \sigma_{e} - Y$$

## Anti-optimization for selecting test conditions

- What test will give us maximum difference between maximum principal stress and maximum principal strain criteria?
- Obviously  $\sigma_1 = \sigma_2 = \sigma_3 = \sigma$
- With max principal stress  $\sigma_e = \sigma$
- With max principal strain  $\sigma_e = \sigma(1-2v) = 0.4$
- Alternatively  $\sigma_1 = -\sigma_2 = -\sigma_3 = \sigma$
- Max principal strain  $\sigma_e = \sigma(1+2\nu) = 1.6$
- What is bad about these test conditions?

#### In plane stress

#### • Figure 4.8



### Strain energy density criterion (Eugenio Beltrami 1835-1900)

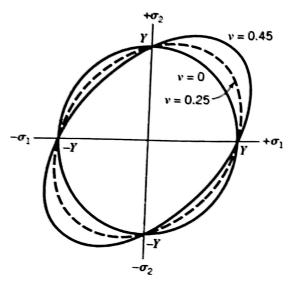
Strain energy density

$$\begin{aligned} &U_0 = \frac{1}{2E} \left[ \sigma_1^2 + \sigma_2^2 + \sigma_3^2 - 2\nu \left( \sigma_1 \sigma_2 + \sigma_1 \sigma_3 + \sigma_2 \sigma_3 \right) \right] \\ &\text{Uniaxial test}: \ \sigma_1 = Y \quad \sigma_2 = \sigma_3 = 0 \qquad U_0 = \frac{1}{2E} \left[ \sigma_1^2 \right] = \frac{Y^2}{2E} \end{aligned}$$

- Criterion  $\sigma_1^2 + \sigma_2^2 + \sigma_3^2 2\nu(\sigma_1\sigma_2 + \sigma_1\sigma_3 + \sigma_2\sigma_3) Y^2 = 0$
- Effective stress  $f = (\sigma_e)^2 Y^2$   $\sigma_e = \sqrt{\sigma_1^2 + \sigma_2^2 + \sigma_3^2 2\nu(\sigma_1\sigma_2 + \sigma_1\sigma_3 + \sigma_2\sigma_3)}$
- Extreme  $\sigma_1 = -\sigma_2 = -\sigma_3 = \sigma$   $\sigma_e = \sigma \sqrt{3 + 2\nu} = 1.90$

#### Plane stress

Depends on Poisson's ratio



**FIGURE 4.9** Strain-energy density yield surface for biaxial stress state ( $\sigma_3 = 0$ ).

What is common to strain and energy criteria?

#### 4.4. Yielding of ductile metals

- Maximum shear-stress criterion (Henri Eduard Tresca, 1814-1885)
- Uniaxial loading  $\sigma_1 = Y; \sigma_2 = 0; \sigma_3 = 0 \rightarrow \tau_{max} = \frac{Y 0}{2} = \frac{Y}{2}$
- Criterion  $f = \sigma_e \frac{Y}{2}$
- · Effective stress

$$\sigma_{e} = \tau_{max} = \max\left(\tau_{1}, \tau_{2}, \tau_{3}\right) \quad \tau_{1} = \frac{\left|\sigma_{2} - \sigma_{3}\right|}{2} \quad \tau_{2} = \frac{\left|\sigma_{3} - \sigma_{1}\right|}{2} \quad \tau_{3} = \frac{\left|\sigma_{1} - \sigma_{2}\right|}{2}$$

- · Conservative for metals in shear
- Extreme case  $\sigma_1 = -\sigma_2 = -\sigma_3 = \sigma$   $\sigma_e = \sigma$

#### Distortional energy density criterion (Ludwig von Mises 1881-1973)

Strain energy density

$$U_{0} = U_{V} + U_{D} = \frac{1}{2E} \left[ \sigma_{1}^{2} + \sigma_{2}^{2} + \sigma_{3}^{2} - 2v \left( \sigma_{1} \sigma_{2} + \sigma_{1} \sigma_{3} + \sigma_{2} \sigma_{3} \right) \right]$$

$$U_{V} = \frac{\left( \sigma_{1} + \sigma_{2} + \sigma_{3} \right)^{2}}{18K} \qquad U_{D} = \frac{\left( \sigma_{1} - \sigma_{2} \right)^{2} + \left( \sigma_{2} - \sigma_{3} \right)^{2} + \left( \sigma_{3} - \sigma_{1} \right)^{2}}{12G}$$

Uniaxial test

$$\sigma_1 = Y$$
  $\sigma_2 = 0$   $\sigma_3 = 0$ 

$$U_D = \frac{(Y-0)^2 + (0-0)^2 + (0-Y)^2}{12G} = \frac{Y^2}{6G}$$

Criterion

Max difference?

$$U_{D} = \frac{12G}{12G} = \frac{1}{6G}$$

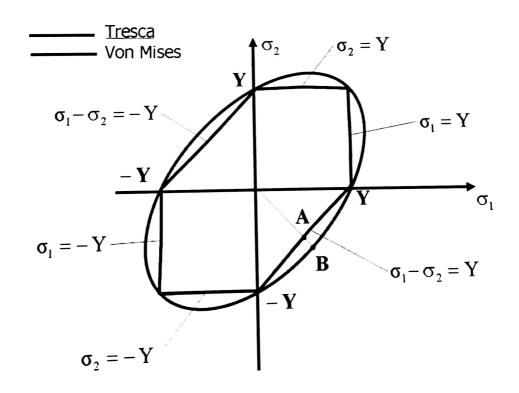
$$\frac{(\sigma_{1} - \sigma_{2})^{2} + (\sigma_{2} - \sigma_{3})^{2} + (\sigma_{3} - \sigma_{1})^{2}}{12G} = \frac{Y^{2}}{6G}$$

$$\sigma_{e} = \sqrt{0.5 \left[ (\sigma_{1} - \sigma_{2})^{2} + (\sigma_{2} - \sigma_{3})^{2} + (\sigma_{3} - \sigma_{1})^{2} \right]}$$

$$f = \sigma_{e} - Y$$

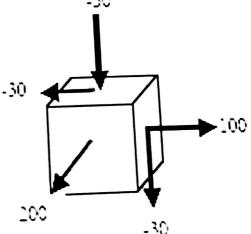
#### In plane stress

• Where is maximum difference?



## Example comapring failure criteria

Stress analysis of a spacecraft structural member gives the state of stress as shown below. If the part is made from an alloy with Y = 500 MPa, check yielding according to Rankine, Tresca and von Mises criteria. What is its safety factor for each criterion?



## Maximum principal stress Criterion (Rankine)

$$\sigma = \begin{bmatrix} 200 & 0 & 0 \\ 0 & 100 & -30 \\ 0 & -30 & -50 \end{bmatrix}$$
 MPa

The principal stresses are:

$$\sigma_1 = 200; \quad \sigma_2 = 105.77; \quad \sigma_3 = -55.77;$$

- Maximum principal stress is 200MPa
- Factor of safety FS=500/200=2.5

# Maximum shear stress Criterion (Tresca)

• Yield function  $f = \sigma_e - \frac{Y}{2}$ 

• Maximum shear stress  $\sigma_c = \frac{|\sigma_1 - \sigma_3|}{2} = \frac{200 + 55.78}{2} = 127.89 \text{ MPa}$ 

· Shear stress for uniaxial tension

$$\frac{Y}{2}$$
 = 250 MPa  $f = 127.89 - 250 < 0$ 

Factor of safety FS=250/127.89=1.95

#### Distortional Energy Density Criterion (von Mises)

Yield function

$$f = \frac{1}{\sqrt{2}} \sqrt{(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2} - Y = 224 - 500$$

- Factor of safety FS=500/224=2.2
- Why are all the results so close?



$$\sigma_1 = 200; \quad \sigma_2 = 105.77; \quad \sigma_3 = -55.77;$$

#### Other yield criteria

- For isotropic materials there is usually substantial difference between yield in tension and compression. Why?
- For orthotropic materials there is also differences between the behavior along the three principal directions and between shear and normal stresses
- Fortunately, for most applications we have at least transverse isotropy

#### Reading assignment

Sections 4.4-5: Question: Under what conditions will we have maximum differences between Tresca and Von-Mises criteria? (consider both the case that Tresca is more conservative and the case that Von Mises is)



Source: www.library.veryhelpful.co.uk/ Page11.htm